

## Review of Compressible Flow

What is a compressible flow?

## Incompressible Flow

- In continuum fluid mechanics, often start by considering conservation/transport equations, e.g.,
  - mass conservation (“continuity equation”)
  - momentum conservation
- For *incompressible flow*, these two conservation equations are sufficient to define the behavior of a fluid
  - but this requires an isodensity constraint ( $\rho = \text{constant}$ )
    - e.g., leads to Bernoulli’s equation (*only valid for incompressible flow along a streamline*)
      - $p_o = p + \frac{1}{2} \rho u^2$  *pressure and velocity (changes) in a flow are linked*
    - assuming constant density is reasonable for nearly incompressible fluid, e.g., many liquids

## Gas Compressibility

- What about a gas flow?
- Gases are **definitely compressible**
  - $\rho = \rho(p, T)$
  - compressibility coefficients for near-tpg (thermally perfect gas) at SATP
$$\alpha = -\left.\frac{1}{\rho} \frac{\partial \rho}{\partial T}\right|_p \sim -O\left(3 \times 10^{-3} \frac{1}{K}\right); \kappa = \left.\frac{1}{\rho} \frac{\partial \rho}{\partial p}\right|_T \sim O\left(10^{-5} \frac{m^2}{N}\right)$$
  - so 10K increase in  $T$  will lower density by  $\sim 3\%$ , as will a 300 Pa drop in  $p$
- When can we treat a gas **flow** as incompressible?
  - when a fluid element only undergoes small pressure and temperature changes as it travels along the flow!!

## Compressible Flow

- Why would  $T$  change in a flow?
  - one reason: heating ( $Q$ )
  - another reason: conversion of flow kinetic energy ( $\frac{1}{2} u^2$ ) to/from thermal energy: **internal energy**( $e$ )/**enthalpy**( $h$ )
- We will **define a compressible flow** as one where the changes in kinetic energy have a non-negligible impact on temperature, and thus density (and pressure)
- To model/analyze compressible flow
  - must include energy conservation equation (in addition to mass and momentum conservation)



## Review of Compressible Flow

What are the governing equations?

## Compressible Flow Equations

- To understand and model compressible flow, we need
  - 1. Conservation equations** for open systems  
(also called transport equations)
    - mass, momentum, **and** energy
    - also species (if reacting)
    - can include diffusion processes  
or if neglect them called “inviscid” versions
  - 2. State equations** (if equilibrium conditions apply)
    - $p$ - $v$ - $T$  relation (leads to compressibility coefficients)
    - caloric relations (give specific heats,  $c_v$  and  $c_p$ )
    - entropic (can get from  $p$ - $v$ - $T$  and caloric)

## Inviscid Conservation/Transport Eqns.

- Mass/continuity**  $\frac{D\rho}{Dt} + \rho \frac{\partial u_j}{\partial x_j} = 0$ 
 $\frac{D?}{Dt} = \frac{\partial ?}{\partial t} + u_j \frac{\partial ?}{\partial x_j}$  substantial derivative
- Momentum**  $\rho \frac{Du_j}{Dt} = -\frac{\partial p}{\partial x_j} + \rho F_j$  body force
- Energy**  $\rho \frac{Dh_o}{Dt} = \frac{\partial p}{\partial t} + \rho F_j u_j + \dot{q}$  volumetric heating, e.g., radiation  
 stagnation enthalpy,  $h_o \equiv h_{mix} + \frac{1}{2} u^2$
- Species**  $\frac{DY_i}{Dt} = \frac{\dot{w}_i}{\rho}$  net mass production rate of species i (I.B.4)

all molecular diffusion terms neglected in these equations

## State Equations - Examples

- p-v-T**  $p/\rho = RT$  thermally perfect (ideal) gas - TPG  
 $(p + a\rho^2)(1/\rho - b) = RT$  Van der Waals state eq: a,b constants
- Caloric**  $e, h$  with  $h = e + pv = e + p/\rho$ 
  - single (or non-reacting) tpg  $e = e(T); h = e(T) + RT$   
 $c_v = de/dT$   $c_p = c_v + R$
  - if reacting tpg,  $R \neq \text{const.}$   $h_{mix} = \sum Y_i h_i = \sum Y_i e_i + \sum (Y_i R_i) T$   
 $= e_{mix}(p, T) + R_{mix}(p, T) T$   
 $= h_{mix}(p, T)$  if non-reacting tpg
- Speed sound**  $a^2 = \gamma (\partial p / \partial \rho)_T$   $a^2 = \gamma RT$  (I.B.5)

## State Equations - Examples

- **Entropic**  $Tds = dh - vdp$ 
    - if tpg  $ds = c_p dT/T - R dp/p$
    - if also cpg  $\Delta s_{12} = c_p \ln(T_2/T_1) - R \ln(p_2/p_1)$
    - so if tpg/cpg fluid element undergoes isentropic process
 
$$\ln(p_2/p_1)^R = \ln(T_2/T_1)^{c_p}$$

$$p_2/p_1 = (T_2/T_1)^{c_p/R} = (T_2/T_1)^{\gamma/(\gamma-1)}$$
 with  $p = \rho RT$   $\rho_2/\rho_1 = (T_2/T_1)^{c_p/R} = (T_2/T_1)^{1/(\gamma-1)}$
- $\frac{c_p}{R} > \frac{c_v}{R} > 1$  **changing  $T$  in isentropic flow will also change  $p$  in same direction (and by more) will also change  $\rho$  in same direction (by more than  $T$  but less than  $p$ )**

## Stagnation Properties

- Properties that would (theoretically) be achieved if a fluid element was brought to rest *in a chosen reference frame*:
  - 1) with no external work,
  - 2) adiabatically + 3) *reversibly?*
- **Stagnation Temperature**  $c_p(T_o - T) = \frac{u^2}{2} \Rightarrow \frac{T_o}{T} = 1 + \frac{u^2/2}{c_p T} = \frac{1}{c_p T} = \frac{\gamma-1}{\gamma RT}$ 

expressions below for tpg+cpg Bulk KE

– from energy conservation: *no work but flow work and adiabatic* **(I.B.6)**  $\frac{T_o}{T} = 1 + \frac{\gamma-1}{2} M^2$

$\Rightarrow T_o$  (and  $h_o$ ) **constant for adiabatic flow**  $M^2 \propto$  kinetic/thermal energy of flow
- **Stagnation Pressure**  $p_o/p = (T_o/T)^{\gamma/(\gamma-1)}$  **(I.B.7)**  $\frac{p_o}{p} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{\gamma/(\gamma-1)}$ 

– for adiabatic + **reversible** **(I.B.7)**  $\frac{p_o}{p} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{\gamma/(\gamma-1)}$

**=isentropic process ( $\Delta s=0$ )**

$\Rightarrow p_o$  (and  $s_o$ ) **constant if also reversible**
- **Stagnation Density**  $\rho_o/p = (T_o/T)^{1/(\gamma-1)}$  **(I.B.8)**  $\frac{\rho_o}{\rho} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{1/(\gamma-1)}$ 

$\rho = p/RT$

## Adiabatic Flow Ellipse

- One way to look at  $M$  effects on flow properties
- **Energy equation**  

$$h_o = h + \frac{u^2}{2} = \text{const}$$
- **Stagnation  $T_o$  also constant**



adiabatic/no work stream tube

**tpg/cpg**  $T_o = T + \frac{\gamma-1}{2} \frac{u^2}{\gamma R} = \text{const}$

$$\frac{2}{\gamma-1} \gamma R T + u^2 = \text{const}$$

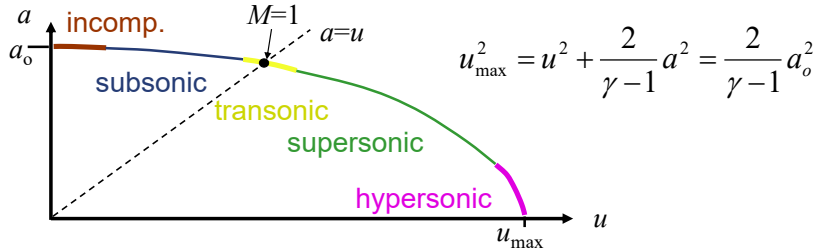
Stagnation speed of sound (no kinetic energy left,  $u=0$ )

$$\frac{2}{\gamma-1} a^2 + u^2 = u_{\text{max}}^2 = \frac{2}{\gamma-1} a_o^2$$

Maximum velocity possible (no thermal energy left,  $T=0$ )

## Adiabatic Flow Ellipse: Flow Regimes

- Transition from low speed ( $a_o$ ) to high speed ( $u_{\text{max}}$ )



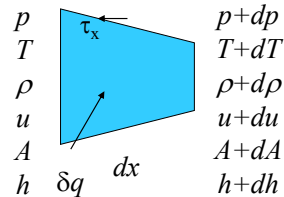
Regime	Description/Interpretation
incomp.	$u \ll a, da \ll du$ , little change in $a(T)$
subsonic	$u \leq a$ , $M$ changes primarily to changes in $u$
transonic	$ u-a  \ll u, a$
supersonic	$u > a$ , $M$ changes through substantial changes in $u$ and $a(T)$
hypersonic	$u \gg a, du \ll da$ , $M$ change mostly due to $a(T)$ changes

## Review of Compressible Flow

How do the compressible flow equations simplify for quasi-1D flow for a tpg/cpg?

## Quasi-1D, Steady Equations

- Examine control volume with differential length  $dx$ , in steady, nonreacting flow, no body forces
  - allow heat transfer and shear stress
  - quasi-1D**  $\equiv$  variations only in  $x$  direction



**Mass** 
$$\frac{d\rho}{\rho} + \frac{1}{2} \frac{du^2}{u^2} + \frac{dA}{A} = 0$$

valid only for  $dA/dx$  small

**Momentum** 
$$\frac{\tau_x}{p} \frac{L_p}{A} dx + \frac{dp}{p} + \frac{1}{2} \frac{u^2}{p/\rho} \frac{du^2}{u^2} = 0$$

shear stress    normal stress    momentum change

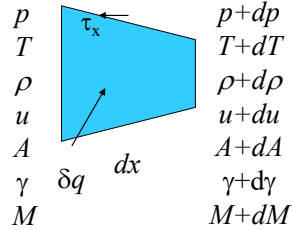
**Energy** 
$$\frac{\delta q}{RT} - \frac{1}{2} \frac{u^2}{RT} \frac{du^2}{u^2} - \frac{dh}{RT} = 0$$

heat addition    KE change    thermal energy change



## Compressible Ideal Gas Equations

- In addition, limit to non-reacting **tpg** (non-react.  $\Rightarrow R = \text{const.}$ )



**Mass**  $\frac{d\rho}{\rho} + \frac{1}{2} \frac{du^2}{u^2} + \frac{dA}{A} = 0$

**Momentum**  $\frac{\tau_x}{p} \frac{L_p}{A} dx + \frac{dp}{p} + \frac{\gamma}{2} M^2 \frac{du^2}{u^2} = 0$

**Energy**  $\frac{\delta q}{c_p T} - \frac{(\gamma-1)}{2} M^2 \frac{du^2}{u^2} - \frac{dT}{T} = 0$

**Ideal Gas Eq. State**  $\frac{dp}{p} - \frac{d\rho}{\rho} - \frac{dT}{T} = 0$

**Mach Number**  $\frac{dM^2}{M^2} - \frac{du^2}{u^2} + \frac{dT}{T} + \frac{d\gamma}{\gamma} = 0$

## Mach Number Changes

- Combine conservation/state equations
  - can algebraically show if **tpg+cpg**  $f = \frac{4\tau_x}{1/2\rho v^2} D = \frac{4A}{L_p}$  friction factor from  $\tau_x$

(I.B.9) 
$$\frac{dM^2}{M^2} = \frac{1 + \frac{\gamma-1}{2} M^2}{1 - M^2} \left\{ \frac{\delta q}{c_p T_o} (1 + \gamma M^2) + \gamma M^2 \frac{f dx}{D} - 2 \frac{dA}{A} \right\}$$

- So we have three ways to change  $M$  of flow
  - **area change** ( $dA$ ): e.g., converging-diverging nozzles
  - **friction**:  $f > 0$ , same effect as  $-dA$
  - **heat transfer**: heating,  $\delta q > 0$ , like  $-dA$   
cooling,  $\delta q < 0$ , like  $+dA$

## Mach Number Changes

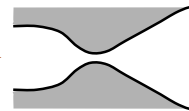
$$\frac{dM^2}{M^2} = \frac{1 + \frac{\gamma-1}{2}M^2}{1-M^2} \left\{ \frac{\delta q}{c_p T_o} (1 + \gamma M^2) + \gamma M^2 \frac{f dx}{D} - 2 \frac{dA}{A} \right\}$$

- **Subsonic flow ( $M < 1$ ):**  $1 - M^2 > 0$ 
  - friction, heating, converging area  $\Rightarrow$  increase  $M$  ( $dM > 0$ )
  - cooling, diverging area  $\Rightarrow$  decrease  $M$  ( $dM < 0$ )
- **Supersonic flow ( $M > 1$ ):**  $1 - M^2 < 0$ 
  - friction, heating, converging area  $\Rightarrow$  decrease  $M$  ( $dM < 0$ )
  - cooling, diverging area  $\Rightarrow$  increase  $M$  ( $dM > 0$ )

## Sonic Condition: Throat Requirement

$$\frac{dM^2}{M^2} = \frac{1 + \frac{\gamma-1}{2}M^2}{1-M^2} \left\{ \frac{\delta q}{c_p T_o} (1 + \gamma M^2) + \gamma M^2 \frac{f dx}{D} - 2 \frac{dA}{A} \right\}$$

- Consider smooth transition subsonic  $\leftrightarrow$  supersonic flow
  - must go through  $M=1$  (sonic condition, denoted by “\*”)
- As  $M \rightarrow 1$ ,  $1 - M^2 \rightarrow 0$ , need  $\{ \}$  term to approach 0
- For **isentropic flow**,
  - sonic condition is  $dA=0$ ,  $A^*$  is a **minimum area (“throat”)** in the flow
- With friction and/or heating, need  $dA > 0$ 
  - $A^*$  occurs in a diverging area region
- For cooling (no friction), need  $dA < 0$ 
  - $A^*$  occurs in a converging area region



## Mach Number Relations

- For tpg/cpg + steady, get equations for each TD property change as function of *Mach number* change

$$\frac{dT_o}{T_o} = \frac{\delta q}{c_p T_o} \quad \frac{d\rho}{\rho} = -\frac{1}{2} \frac{du^2}{u^2} - \frac{dA}{A} \quad \frac{dp}{p} = -\frac{\gamma}{2} M^2 \left( \frac{du^2}{u^2} + \frac{fdx}{D} \right)$$

$$\frac{dT}{T} = \frac{dT_o}{T_o} - \frac{\frac{\gamma-1}{2} M^2}{1 + \frac{\gamma-1}{2} M^2} \frac{dM^2}{M^2} \quad \frac{dp_o}{p_o} = \frac{dp}{p} + \frac{\frac{\gamma}{2} M^2}{1 + \frac{\gamma-1}{2} M^2} \frac{dM^2}{M^2}$$

$$\frac{du^2}{u^2} = \frac{dT_o}{T_o} + \frac{\frac{2}{1}}{1 + \frac{\gamma-1}{2} M^2} \frac{dM^2}{M^2} \quad \frac{ds}{R} = \frac{\gamma}{\gamma-1} \frac{dT_o}{T_o} - \frac{dp_o}{p_o}$$

- Analytic solutions to these differential eqns. exist if **ONLY**
  - area change : **Isentropic** (e.g., nozzle) flow
  - OR friction : **Fanno** flow
  - OR heat transfer : **Rayleigh** flow

## Review of Compressible Flow

How does quasi-1D, steady isentropic flow behave for a tpg/cpg?

## Isentropic Flow: Property Changes

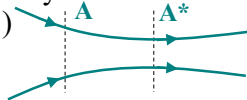
- For tpg/cpg,  $T_o$  and  $p_o$  (and  $\rho_o$ ) are constant in isentropic flow

$$\frac{T_o}{T} = 1 + \frac{\gamma-1}{2} M^2 < \frac{\rho_o}{\rho} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{1/\gamma-1} < \frac{p_o}{p} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{\gamma/\gamma-1}$$

- If  $M$  increases (an **expansion**)
  - $T$ ,  $\rho$ , and  $p$  will decrease
- If  $M$  decreases (a **compression**)
  - $T$ ,  $\rho$ , and  $p$  will increase
- $T$  will change least,  $p$  will change most

## Area Ratio

- For isentropic flow, look at effect of area change on  $M$  by comparing  $A$  at any point to area at sonic point ( $A^*$ )
  - \* refers to properties of a flow if isentropically accel./decel. to  $M=1$  sonic (e.g.  $\rho^*$ ,  $T^*$ , ...)
  - alternative to stagnation as ref. state



- Use **mass conservation** to find relation

$$\rho u A = \rho^* u^* A^* \quad \frac{1}{M}$$

$$\frac{A}{A^*} = \frac{\rho^* u^*}{\rho u} = \frac{\rho^* \rho_o a^*}{\rho \rho_o a} \frac{1}{M}$$

$$\frac{\rho_o}{\rho} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{1/\gamma-1}$$

$$\frac{\rho_o}{\rho^*} = \left(1 + \frac{\gamma-1}{2}\right)^{1/\gamma-1}$$

$$\frac{A}{A^*} = \frac{1}{M} \left\{ \frac{1 + \frac{\gamma-1}{2} M^2}{\frac{\gamma+1}{2}} \right\}^{\gamma+1/2(\gamma-1)}$$

$$\frac{a^*}{a} = \sqrt{\frac{T^*}{T}} = \sqrt{\frac{T^* T_o}{T T_o}} \quad \text{with} \quad \frac{T_o}{T} = 1 + \frac{\gamma-1}{2} M^2$$

$$\frac{T_o}{T^*} = 1 + \frac{\gamma-1}{2}$$

(I.B.10)  
for tpg/cpg

## Area Ratio Results

- Two (isentropic) solutions for a given  $A/A^*$

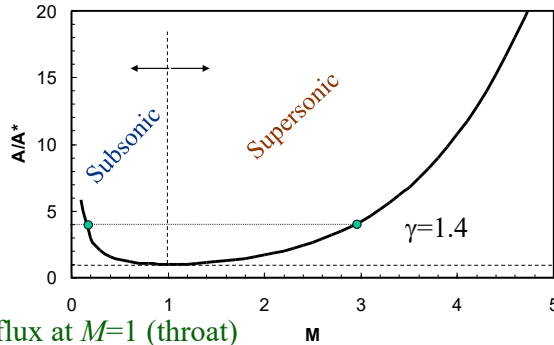
- one **subsonic**
- one **supersonic**

- $A \geq A^*$  always
- Acceleration to high  $M$  requires large  $A/A^*$

- $\frac{\dot{m}/A}{(\dot{m}/A)^*} = \frac{\rho u}{\rho^* u^*} = \frac{A^*}{A}$

- maximum mass flux at  $M=1$  (throat)

$$\frac{A}{A^*} = \frac{1}{M} \left\{ \frac{1 + \frac{\gamma-1}{2} M^2}{\frac{\gamma+1}{2}} \right\}^{\gamma+1/2(\gamma-1)}$$



## Mass Flux and Stagnation Properties

- Examine mass flux in terms of stagnation conditions

$$\frac{\dot{m}}{A} = \rho u = \frac{p}{RT} Ma = \frac{p}{RT} M \sqrt{\gamma RT} \quad \leftarrow \text{tpg}$$

(I.B.11)

$$\frac{\dot{m}}{A} = \frac{p_o}{\sqrt{RT_o}} \frac{\sqrt{\gamma} M}{\left(1 + \frac{\gamma-1}{2} M^2\right)^{\gamma+1/2(\gamma-1)}}$$

tpg, cpg

$$\frac{T_o}{T} = 1 + \frac{\gamma-1}{2} M^2$$

$$\frac{p_o}{p} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{\gamma/\gamma-1}$$

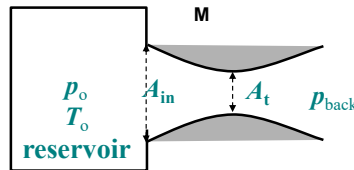
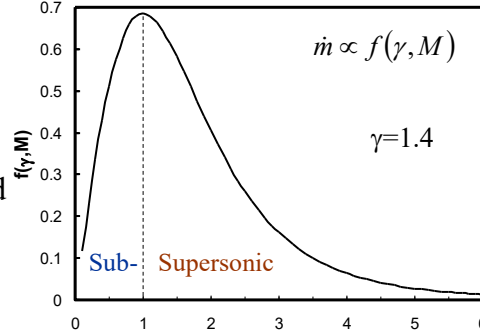
OR  $\frac{\dot{m}}{A} = \frac{p_o}{\sqrt{RT_o}} f(\gamma, M)$

for given isentropic flow, all stagnation (and sonic) properties constant, so is mass flow rate

## Choked Flow

$$\dot{m} = A \frac{P_o}{\sqrt{RT_o}} f(\gamma, M)$$

- For **fixed stagnation properties** and flow area
    - max  $\dot{m}$  at  $M = 1$
  - For nozzle with **fixed stagnation properties** and **sonic throat**
    - can't alter mass flow rate by changing downstream pressure boundary condition (*back pressure*)
- ⇒ **Choked Flow**



## Choked Mass Flowrate

- Sonic condition (*at throat*) ⇒ maximum flow rate

$$\dot{m}_{\max} = A^* \frac{P_o}{\sqrt{RT_o}} \underbrace{\sqrt{\gamma \left(1 + \frac{\gamma-1}{2}\right)^{\frac{\gamma+1}{2(1-\gamma)}}}}_{f(\gamma, 1)} \quad \text{(I.B.12) for tpg/cpg}$$

- To **increase** mass flowrate
  - increase  $A^*$  ( $=A_t$  throat size)
  - increase  $P_o$  and/or decrease  $T_o$  (either increases  $\rho_o$ )
- $f(\gamma, 1)$  typically near 0.7

$$\dot{m}_{\max} \approx 0.7 \frac{P_o}{\sqrt{RT_o}} A_{throat} \quad f(\gamma, 1) = \begin{cases} 0.726 & \gamma = 5/3 \\ 0.685 & \gamma = 1.4 \\ 0.667 & \gamma = 1.3 \end{cases}$$

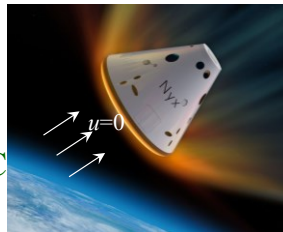
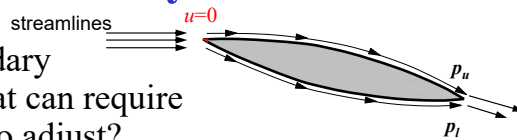
“rule of thumb” for **choked** gas flows

# Review of Compressible Flow

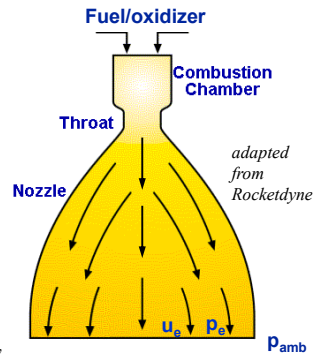
How does a supersonic flow adjust to boundary conditions?

# Supersonic Boundary Conditions

- What are the boundary conditions (BC) that can require a supersonic flow to adjust?
  - surface requiring change in flow direction (“turn”)
  - stagnation ( $u=0$ )
  - pressure matching
- So 2 kinds BC
  - velocity,
  - pressure



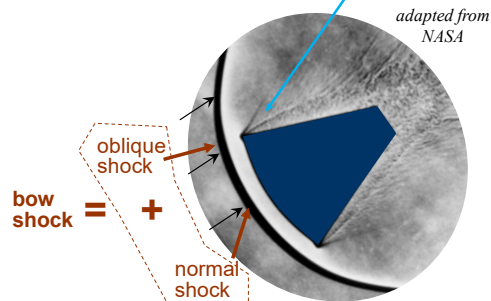
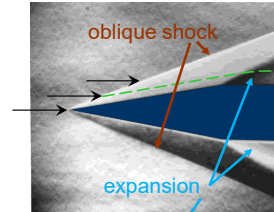
adapted from The Exploration Company



adapted from Rocketdyne

## Waves

- How do supersonic flows adapt to changes in boundary conditions (BC)?
- Through waves that propagate BC into supersonic flow
  - “strong waves”  
**shocks**
    - *compression only*
  - “weak waves”  
**sound waves**
    - *expansion or compression*



Compressible Flow -31  
Copyright © 2001, 2003, 2007, 2015, 2019, 2023  
by Jerry M. Seitzman. All rights reserved.

**AE6050**

## Review of Waves

- From this point on, restrict analysis to flows that are
  - steady
  - adiabatic
  - inviscid (except within shocks)
  - 1-d or 2-d
- Review
  - normal shocks
  - oblique shocks
  - Prandtl-Meyer expansions and compressions

Compressible Flow -32  
Copyright © 2001, 2003, 2007, 2015, 2019, 2023  
by Jerry M. Seitzman. All rights reserved.

**AE6050**



## Review of Compressible Flow

What is a normal shock and how do flow properties change?

## Sound Waves vs Shock Waves

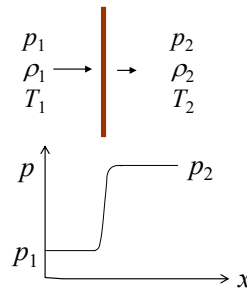
- **Sound waves**

- weak,  $d\rho/\rho \rightarrow 0$
- reversible and isentropic

$$\begin{array}{c} p \\ \rho \\ T \end{array} \rightarrow \left| \begin{array}{c} p+dp \\ \rho+d\rho \\ T+dT \end{array} \right.$$

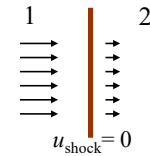
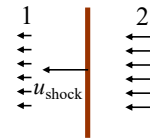
- **Shocks**

- *strong compressions*,  $\rho_2 > \rho_1$
- “thin” regions ( $\sim \mu\text{m}$ 's at STP)
  - changes in fluid properties are *nearly discontinuous*
- rapid change in properties due to internal viscous stresses
  - so *irreversible*
- excluding radiation, *adiabatic*
  - so **nonisentropic**



## Normal Shock Waves

- For **normal shock**
  - wave perpendicular to flow (propagation) direction
- Shock is nonequilibrium process internally, but assume
  - flow *before shock* (1) is **in equilibrium**
  - flow *after shock* (2) is **in equilibrium**
- Easiest way to analyze shock is with control volume in shock's reference frame (moving with shock)
  - equations first studied by Rankine (~1870) and Hugoniot (~1877)



## Governing Equations

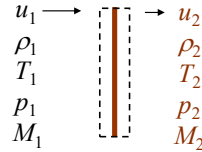
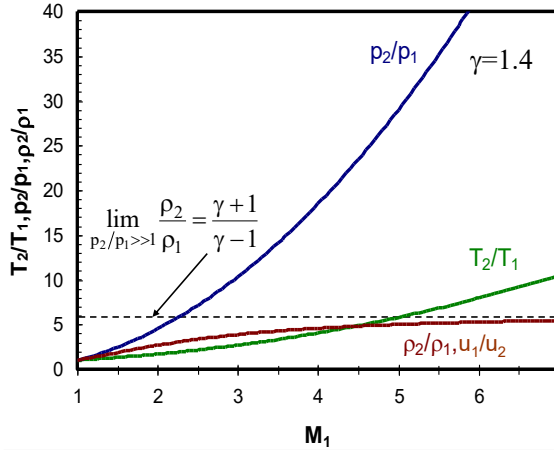
- Conservation equations become
 

		$u_1$	→	$u_2$
		$p_1$		$p_2$
		$\rho_1$		$\rho_2$
		$T_1$		$T_2$
		$h_1$		$h_2$
		$a_1$		$a_2$

  - mass  $\dot{m}/A = \rho_1 u_1 = \rho_2 u_2$
  - momentum  $p_1 A_1 + \dot{m} u_1 = p_2 A_2 + \dot{m} u_2$   
 $p_1 + \rho_1 u_1^2 = p_2 + \rho_2 u_2^2 = \text{constant}$
  - energy  $h_1 + u_1^2/2 = h_2 + u_2^2/2 = h_o = \text{constant}$
  - combine  $h_2 - h_1 = \frac{1}{2} (p_2 - p_1) (1/\rho_1 + 1/\rho_2)$  **Shock Hugoniot Equation (I.B.13)**
- For **tpg/cpg**  $\frac{\gamma}{\gamma-1} (p_2/\rho_2 + p_1/\rho_1) = \frac{1}{2} (p_2 - p_1) (1/\rho_1 + 1/\rho_2)$ 
  - so density ratio  $\rho_2/\rho_1 = \left(1 + \frac{\gamma+1}{\gamma-1} \frac{p_2}{p_1}\right) / \left(\frac{\gamma+1}{\gamma-1} + \frac{p_2}{p_1}\right)$
  - also  $T_{o2} = T_{o1}$   $p_2/p_1 = [2\gamma/(\gamma+1)] M_1^2 - (\gamma-1)/(\gamma+1)$   
 $M_2^2 = \left(M_1^2 + \frac{2}{\gamma-1}\right) / \left(\frac{2\gamma}{\gamma-1} M_1^2 - 1\right)$   $T_2/T_1 = \left(1 + \frac{\gamma-1}{2} M_1^2\right) / \left(1 + \frac{\gamma-1}{2} M_2^2\right)$

## Normal Shock Property Changes

- Based on tpg/cpg eqn's. using shock fixed reference frame



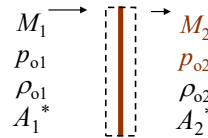
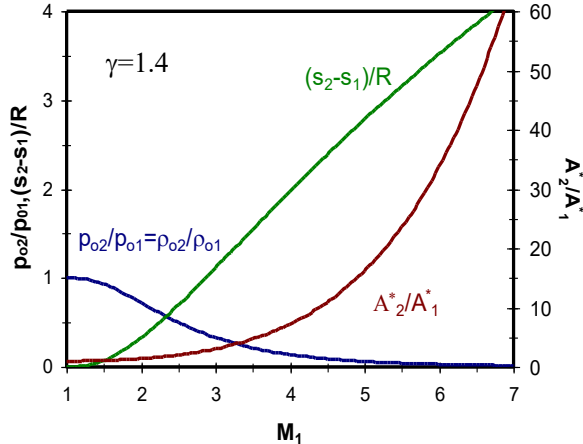
- $T, p$  and  $\rho$  increase,  $u$  decreases
- $p$  increase across normal shock is **greatest static property change**
- Density ratio and velocity ratio approach **limit**
- Also  $M_2 < 1$ , **post-shock flow subsonic**

Compressible Flow -37  
Copyright © 2001, 2003, 2007, 2015, 2019, 2023 by Jerry M. Seitzman. All rights reserved.

Which of these still hold for moving shock? **AE6050**

## Normal Shock Property Changes

- Based on tpg/cpg eqn's. using shock fixed reference frame



- $T_o = \text{constant}$
- $p_o$  and  $\rho_o$  drop
- Entropy increases
- Sonic area increases
  - larger throat required after shock to reach sonic flow (same mass flowrate, lower  $p_o$ )

$$\dot{m} \propto A^* p_o$$

Compressible Flow -38  
Copyright © 2001, 2003, 2007, 2015, 2019, 2023 by Jerry M. Seitzman. All rights reserved.

Which of these still hold for moving shock? **AE6050**

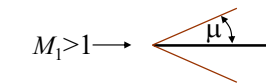
## Review of Compressible Flow

What is an oblique shock and how do flow properties change?

## Oblique Shock Waves

- Recall Mach wave

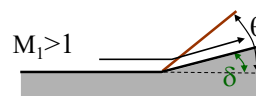
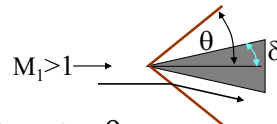
- consider infinitely thin body
- no flow turn required
- infinitesimal wave



$$\mu = \sin^{-1}(1/M_1)$$

- Oblique shock

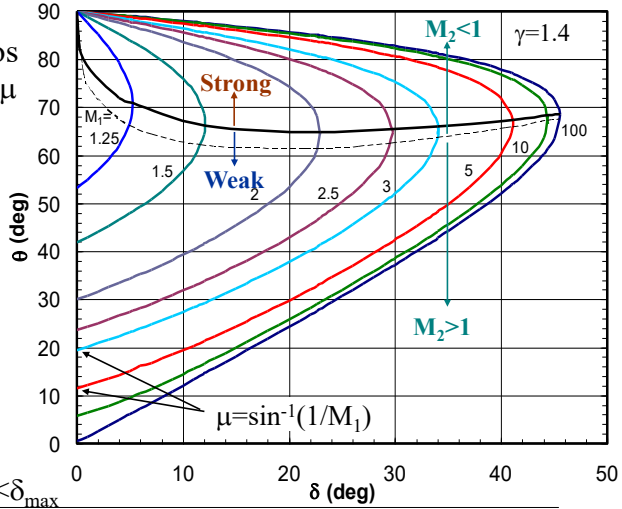
- example, finite-sized wedge, half-angle,  $\delta$ , in supersonic flow
- flow must make sudden, finite turn and compress (flow area reduced)  $\theta > \mu$
- if shock can be attached to tip  $\Rightarrow$  **oblique shock** at angle  $\theta$
- similar for sharp concave corner





## Oblique Shock Chart

- **Weak shocks**
  - lower  $M_{1n}$  ratios
  - smaller  $\theta$ ,  $\theta_{\min} = \mu$
  - usually  $M_2 > 1$
- **Strong shocks**
  - $\theta_{\max} = 90^\circ$  (normal shock)
  - always  $M_2 < 1$
- **Both** for  $\delta = 0$ 
  - no turn for normal shock or Mach wave
- $\delta_{\max}(M_1)$ 
  - oblique shocks only exist for  $\delta < \delta_{\max}$



Compressible Flow -43  
Copyright © 2001, 2003, 2007, 2015, 2019, 2023  
by Jerry M. Seitzman. All rights reserved.

AE6050

## Review of Compressible Flow

What are Prandtl-Meyer fans and how do flow properties change?

Compressible Flow -44  
Copyright © 2001, 2003, 2007, 2015, 2019, 2023  
by Jerry M. Seitzman. All rights reserved.

AE6050

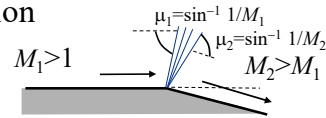
## Expansions and Smooth Compressions

- For oblique shocks, we examined supersonic flow over **sharp** concave corners/turns

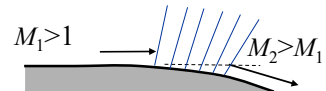
- required sudden finite compression

- What happens if:

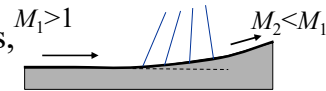
- 1) turn is **convex** (expansion)?



- 2) turn is **gradual**?  
(convex or concave)



- Get series of infinitesimal waves, if isentropic  $\Rightarrow$  **Mach waves**
- **Prandtl-Meyer “fan”**



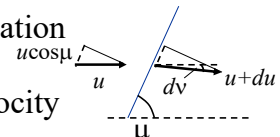
*what happens when weak compression waves converge?*

## Prandtl Meyer Wave

- **Solution approach**

- apply mass and momentum conservation to differential CV (wave aligned)

- result is no change in tangential velocity



$$u \cos \mu = (u + du) \cos (\mu + dv) \Rightarrow \begin{aligned} &= \cos \mu \cos dv - \sin \mu \sin dv \\ &= \cos \mu - dv \sin \mu \end{aligned}$$

$$(I.B.15) \quad \frac{du}{u} = \frac{\sin \mu}{\cos \mu} dv = \frac{\sin^2 \mu}{1 - \sin^2 \mu} dv$$

*Relates velocity change to turn angle*

- for (isentropic) Mach wave, Mach angle given by  $\sin \mu = a/u$

**Prandtl Meyer function**

*for tpg/cpg*

$$\frac{du}{u} = \frac{(a/u)^2}{1 - (a/u)^2} dv \Rightarrow \frac{du}{u} = \frac{1}{M^2 - 1} dv \quad (I.B.16)$$

$$dv = \frac{\sqrt{M^2 - 1}}{1 + \frac{\gamma - 1}{2} M^2} \frac{dM}{M}$$

$\Rightarrow$  turn function of only M for tpg/cpg

## PM Results: tpg/cpg

- To get overall change across PM fan, integrate

$$\delta = v_2 - v_1 = \int dv \quad \text{(I.B.17)}$$

- For tpg/cpg

– using

$$v=0 \text{ at } M=1$$

- Properties changes?

– reversible +

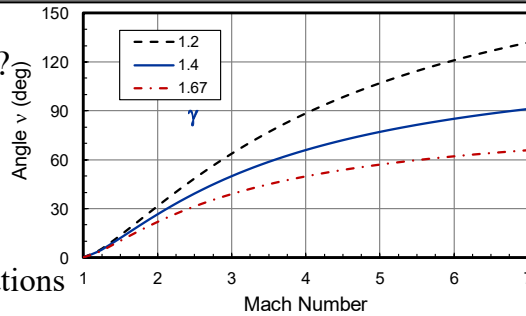
adiabatic

–  $T_o, p_o, \rho_o, s$

constant

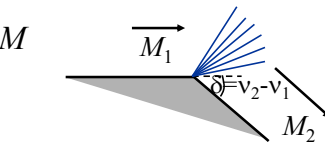
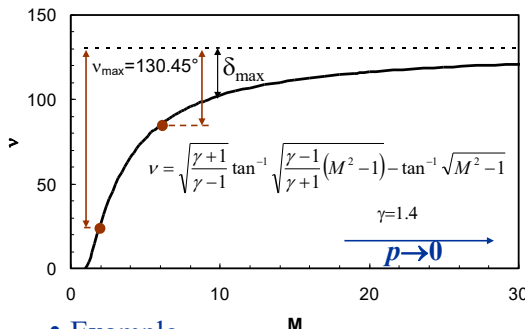
–  $T, p, \rho$ : isen. relations

$$v_2 - v_1 = \left[ \sqrt{\frac{\gamma+1}{\gamma-1}} \tan^{-1} \sqrt{\frac{\gamma-1}{\gamma+1}} (M^2 - 1) - \tan^{-1} \sqrt{M^2 - 1} \right]_{M_1}^{M_2}$$



## Maximum Prandtl Meyer Turn Angle

- Examine plot of  $v$  as function of  $M$

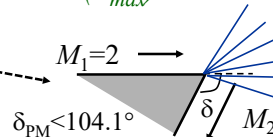


- as  $M$  increases, reach **maximum turn angle** ( $v_{max} \sim 130.5^\circ$  for  $\gamma=1.4$ )
- so as  $M_1$  increases, **why?** max. angle flow can turn ( $\delta_{max}$ ) decreases

- Example

$$M_1 = 2 \Rightarrow \delta_{max} = 130.45^\circ - 26.38^\circ = 104.1^\circ$$

$$M_1 = 6 \Rightarrow \delta_{max} = 130.45^\circ - 84.96^\circ = 45.5^\circ$$



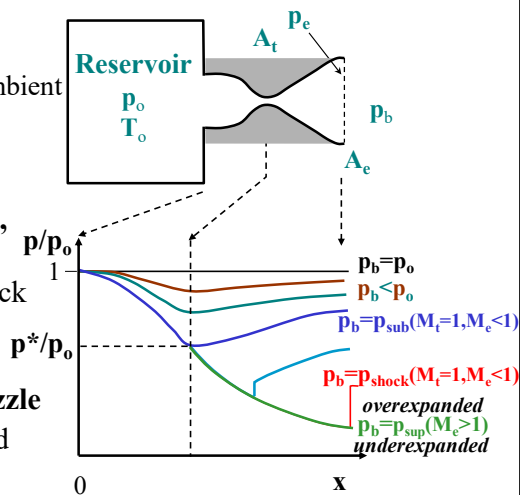


# Review of Compressible Flow

Examples of aerospace applications with supersonic flowfields containing shocks and expansions

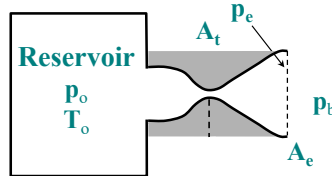
## CD (Laval) Nozzle

- For converging-diverging nozzle
  - back pressure  $p_b$  (e.g., ambient pressure) is downstream boundary condition
- If  $p_b/p_o < p_{sub}/p_o$ , choked flow (**sonic throat**,  $M_t=1$ )
- Lowering  $p_b/p_o$  moves shock from throat to exit
- If  $p_b/p_o < p_{shock}/p_o$ ,  $M_e > 1$  and **isentropic flow in nozzle**
- $p_b = p_{sup}$ , perfectly expanded exhaust



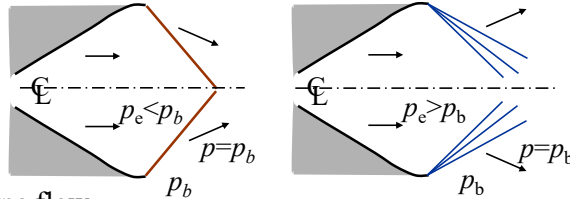
## CD (Laval) Nozzle

- What happens if under- or over-expanded?
- **Underexpanded**
  - pressure BC at exit requires supersonic expansion
  - ⇒ Prandtl-Meyer type flow



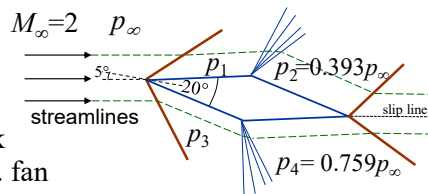
- **Overexpanded**

- pressure BC requires supersonic compression process
- ⇒ oblique shock type flow



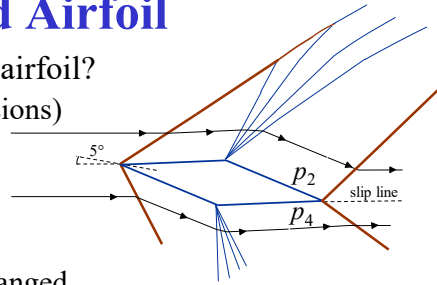
## Diamond Airfoil

- 20° diamond (sharp corners) airfoil in Mach 2 air flow
  - assuming air is tpg/cpg with  $\gamma=1.4$
- **Flow features**
  - upper surface
    - 5° upward = obl. shock
    - 20° down = PM expan. fan
  - lower surface
    - 15° downward = obl. shock
    - 20° up = PM expan. fan
  - at trailing edge, must match  $p$  and velocity direction = 2 obl. shocks in this case (upper stronger)
    - trailing edge flow close to 0° here, not general result



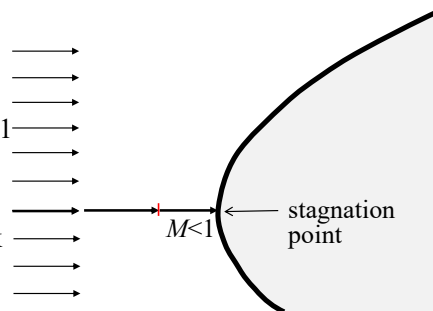
## Diamond Airfoil

- What happens farther from airfoil?
  - **waves** (shocks and expansions) **can interact**
  - expansion waves are carrying information from airfoil that the required turn angle has changed
    - so leading oblique shock gets weaker (curves)
      - shock angle decreases until it becomes a Mach wave
    - PM waves also curve as approaching  $M \uparrow$
    - occurs again as waves from trailing edge propagate away from airfoil



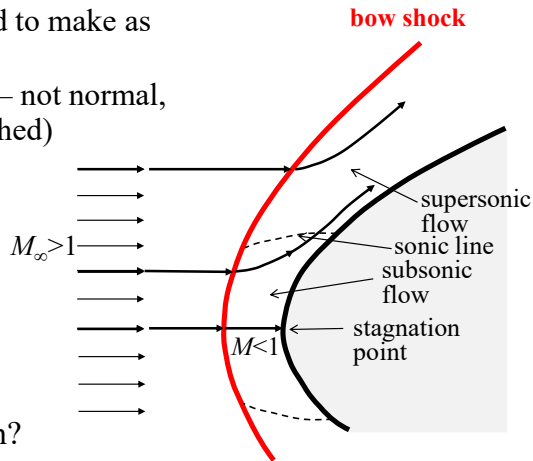
## Blunt Body

- What happens for non-sharp (blunt) body in supersonic flow?
- Example symmetric body, zero angle of attack
- Consider center streamline
  - flow must come to rest at body
  - how can supersonic flow know to slow down?
    - 90° turn too sharp for attached oblique shock
    - can you have attached normal shock?



## Blunt Body Example

- Nearby streamline
  - does flow need to make as sharp a turn?
  - weaker shock – not normal, oblique (detached)
- Rest of flowfield
  - progressively weaker shock required
  - how does shock know it can be weaker in supersonic region?

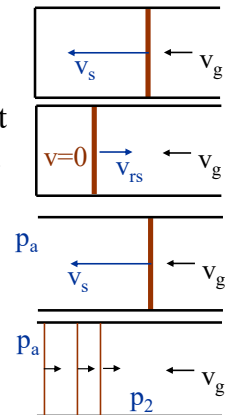


Compressible Flow .56  
Copyright © 2001, 2003, 2007, 2015, 2019, 2023  
by Jerry M. Settlemire. All rights reserved.

AE6050

## Reflected Waves

- What happens when wave “hits” a boundary  $\Rightarrow$  new BC
- Example: moving normal shock
  - if incident shock hits solid wall, get reflected (normal) **shock** - required to satisfy **velocity BC** ( $v=0$ )
  - if it hits open end, get **reflected expansion waves** - satisfy **pressure BC** ( $p=p_a < p_2$ )

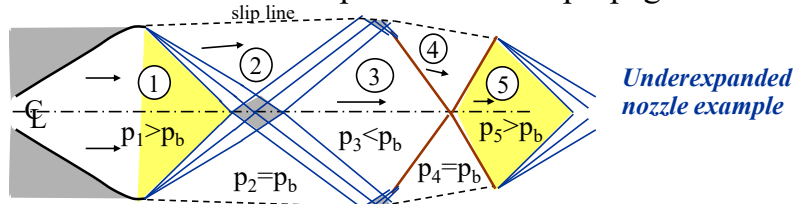


Compressible Flow .56  
Copyright © 2001, 2003, 2007, 2015, 2019, 2023  
by Jerry M. Settlemire. All rights reserved.

AE6050

## Nozzle Exhaust: Reflected Waves

- CD nozzle under/over-expanded  $\Rightarrow$  BCs propagate



- Regions of high pressure, also have high density (and temperature); hot, dense gases emit light (radiation)



SR-71 at takeoff

from NASA